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First steps of the melting of an amorphous polymer through a hot-end of a material extrusion additive manufacturing

Franck Pigeonneau¹, Sarah Marion², Lucas Sardo³ & Thomas Joffre⁴ ¹Mines Paris | PSL Univ. - Centre of Materials Forming, CNRS UMR 7635, Sophia Antipolis, France

²Mines Saint-Etienne, Univ Lyon, CNRS UMR 5307 LGF, Centre SMS, Saint-Etienne, France ³Sciences Computers Consultants, Saint-Etienne, France

⁴IPC Innovation Plasturgie Composites, Bellignat, France



Material extrusion



2

guide heat sink fan heatbreak heat block 10.0 nozzle

Figure 1: Extruder E3D-V6 according to E3D-online¹.

¹E3D-online: E3D Help Centre, https://e3d-online.zendesk.com/hc/en-us, 2021.



2

Material extrusion



Figure 1: Extruder E3D-V6 according to E3D-online¹ and presence of air gap.

¹E3D-online: E3D Help Centre (see n. 1).



2

Material extrusion



Figure 1: Extruder E3D-V6 according to E3D-online¹.

How does the polymer melt?

¹E3D-online: E3D Help Centre (see n. 1).

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3



Figure 2: Melting according to Bellini et al.².

²A. Bellini/S. Güçeri/M. Bertoldi: Liquefier dynamics in fused deposition, in: J. Manuf. Sci. Eng. 126.2 (2004), pp. 237–246.

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4

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Figure 3: Melting according to Osswald et al.³.

³T. A. Osswald/J. Puentes/J. Kattinger: Fused filament fabrication melting model, in: Addit. Manuf. 22 (2018), pp. 51–59.



4

Material extrusion



Figure 3: Melting according to Osswald et al.³.

Numerical computation of the melting of polymer in the extruder

³Osswald/Puentes/Kattinger: Fused filament fabrication melting model (see n. 3).





Figure 4: Channel with dimensions (mm) used in the numerical computation.

Main assumptions

- Fluid dynamics & heat transfer are accounted for in the heatblock channel.
- Polymer considered as solid at the entrance of the channel.
- Two-phase dynamics is considered using a single fluid representation.
- Interface between the two phases is assumed sharp and implicitly tracked by a level-set method.
- The two phases are incompressible.
 - Polymer viscosity follows a Carreau-Yasuda's law.



Table 1: Characteristic variables used in the normalisation.

Table 2: Dimensionless numbers, names, definitions.

Reynolds	capillary	Péclet	Brinkman	Weissenberg
Re	Ca	\mathbf{Pe}	Br	Wi
$\rho_P UD$	$U\eta_{p,0}$	UD	$\eta_{P,0}U^2$	λU
$\eta_{P,0}$	γ	ap	$k_p(T_{\infty}-T_0)$	D





Governing equations

$$\boldsymbol{\nabla}\cdot \mathbf{u} = \mathbf{0}, \tag{4}$$

$$\operatorname{Re}\rho\frac{D\mathbf{u}}{Dt} = -\boldsymbol{\nabla}P + \boldsymbol{\nabla}\cdot[2\eta(\dot{\gamma},\theta,\varphi)\dot{\boldsymbol{\epsilon}}] + \operatorname{Ca}^{-1}\kappa\|\boldsymbol{\nabla}\varphi\|\delta(\varphi)\mathbf{n}_{\varphi}, \qquad (5)$$

$$\operatorname{Pe} \rho C_{\rho} \frac{D\theta}{Dt} = \boldsymbol{\nabla} \cdot [k \boldsymbol{\nabla} \theta] + \operatorname{Br} \eta(\dot{\gamma}, \theta, \varphi) \dot{\gamma}^{2}, \qquad (6)$$

$$\frac{D\varphi}{Dt} = 0. \tag{7}$$





7

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Surface operators at the interface

$$\mathbf{n}_{\varphi} = \frac{\boldsymbol{\nabla}\varphi}{\|\boldsymbol{\nabla}\varphi\|},\tag{8}$$

$$\kappa = -\nabla_{S} \cdot \mathbf{n}_{\varphi}. \tag{9}$$





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Average properties

$$\rho(\varphi) = H(\varphi) + [1 - H(\varphi)] \rho_{\text{air}}, \qquad (10)$$

$$\frac{1}{k(\varphi)} = H(\varphi) + \frac{1 - H(\varphi)}{k_{\text{air}}}. \qquad (11)$$
Cernef



7



Cem

8

1. Problem statement



Figure 5: Boundary conditions on the channel with the initial position of the polymer and air.

2. Numerical implementation

- Finite element method using our own C++ library, CIMLIB_CFD.
- Implicit Euler scheme is implemented to describe time derivative.
- ▶ \mathbb{P}_1 -bubble/ \mathbb{P}_1 element to satisfy the inf sup condition.
- Equations on θ and φ are stabilised using Streamline Upwind/Petrov-Galerkin (SUPG) method.
- Eikonal property of φ , i.e., $\|\nabla \varphi\| = 1$ is conserved using a direct re-initialisation scheme⁴.

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9

► An adaptive anisotropic mesh is used⁵.

⁴M. Shakoor et al.: An efficient and parallel level set reinitialization method - Application to micromechanics and microstructural evolutions, in: Appl. Math. Modell. 39.23 (2015), pp. 7291–7302. ⁵G. Jannoun et al.: Anisotropic meshing with time-stepping control for unsteady convection-dominated problems, in: Appl. Math. Modell. 39.7 (2015), pp. 1899–1916.

3. Materials and working conditions

Table 3: Parameters used in Carreau-Yasuda and Arrhenius laws for the ABS.



Figure 6: $\eta(\dot{\gamma}, T)/a_T$ vs. $\dot{\gamma}a_T$ for the ABS at 4 T.



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3. Materials and working conditions

Table 4: Thermal properties for the ABS⁶ and air⁷.

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11

ρ	C_{ρ}	k			
kg/m^3	$ m Jkg^{-1}K^{-1}$	${ m W}{ m m}^{-1}{ m K}^{-1}$			
	ABS				
1150	2100	0.21			
Air, $T = T_{ext}$					
0.701	1034.78	0.0399			

⁶M. E. Mackay et al.: The performance of the hot end in a plasticating 3D printer, in: J. Rheol. 61.2 (2017), pp. 229–236.

⁷K. Kadoya/N. Matsunaga/A. Nagashima: Viscosity and Thermal Conductivity of Dry Air in the Gaseous Phase, in: J. Phys. Chem. Ref. Data 14.4 (1985), pp. 947–970.

3. Materials and working conditions

Table 5: Definitions of working conditions.

Case	U (m min ⁻¹)	$U_{ m in}~({ m mms}^{-1})$	T_0 (°C)	T_{∞} (°C)
1	1	0.87		
2	3	2.61	24	230
3	6	5.22		

⁶Mackay et al.: The performance of the hot end in a plasticating 3D printer (see n. 6). ⁷Kadoya/Matsunaga/Nagashima: Viscosity and Thermal Conductivity of Dry Air in the Gaseous Phase (see n. 7).

Table 4: Thermal properties for the ABS^6 and air^7 .







Triple line location and T







Figure 7: z_{cl} vs. t (s) for the three cases.



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The feeding force is numerically evaluated using the Berker's theorem⁸: Theorem

For a velocity field with no-slip condition on a wall, the tension on the wall is given by

$$\mathbf{T} = \left(P - \frac{4\eta}{3} \nabla \cdot \mathbf{u}\right) \mathbf{n} + 2\eta \mathbf{n} \times \boldsymbol{\omega},\tag{12}$$

with

$$\boldsymbol{\omega} = \frac{1}{2} \boldsymbol{\nabla} \times \mathbf{u}. \tag{13}$$

Proof.

Corollary of the Stokes theorem.

► Vorticity is determined using Paraview.

⁸R. Berker: Intégration des équations du mouvement d'un fluide visqueux incompressible, in: Handbuch der Physik, Berlin 1963.



Figure 8: Feeding force vs. *t* for $U=1 \text{ m min}^{-1}$.



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Figure 8: Feeding force vs. *t* for $U=1 \text{ m min}^{-1}$.





15



Figure 8: Feeding force vs. t for $U=1 \text{ m min}^{-1}$.



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15



Figure 8: Feeding force vs. t for $U=1 \text{ m min}^{-1}$.





Figure 9: Feeding force vs. t for $U=6 \text{ m min}^{-1}$.



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Figure 9: Feeding force vs. *t* for $U=6 \text{ m min}^{-1}$.



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Figure 9: Feeding force vs. *t* for $U=6 \text{ m min}^{-1}$.



4. Results t=6 s 5.0e+02 480 80 460 440 60 -420 T (K) 400 (N) 40 -380 360 - 340



Figure 9: Feeding force vs. *t* for $U=6 \text{ m min}^{-1}$.



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16

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4. Results

Figure 9: Feeding force vs. t for $U=6 \text{ m min}^{-1}$.





Figure 10: F (N) vs. U_{in} (mm min⁻¹) for T_{∞} =230 °C and comparison with Serdeczny et al.⁹ at T_{∞} =225 °C.

⁹M. P. Serdeczny et al.: Numerical modeling of the polymer flow through the hot-end in filament-based material extrusion additive manufacturing, in: Addit. Manuf. 36 (2020), p. 101454.

5. Synthesis

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- Numerical investigations focus on the melting in a particular extruder (E3D-V6).
- ► Two-phase flow accurately described using a level-set method.
- Air gap persists at high feeding velocity.
- ▶ Dynamics of the rising of the contact line in agreement with Hong et al.¹⁰.
- ► At small feeding rate, polymer fully melted in agreement with Bellini et al. (2004).
- At large feeding rate, polymer partially melted in agreement with Osswald et al. (2008).
- Details can be found in Marion et al.¹¹.

¹⁰Y. Hong et al.: In-situ observation of the extrusion processes of Acrylonitrile Butadiene Styrene and Polylactic Acid for material extrusion additive manufacturing, in: Addit. Manuf. 49 (2022), p. 102507. ¹¹S. Marion et al.: First steps of the melting of an amorphous polymer through a hot-end in Fused Filament Fabrication, in: Addif. Manuf. 65 (2023), p. 103435.

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Sarah Marion Lucas Sardo Thomas Joffre **Thank you for your attention!**

Contact: franck.pigeonneau@minesparis.psl.eu

